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STREGTH OF MATERIAL
OR
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MECHANICS OF SOLIDS
OR
MECHANICS OF STRUCTURE
OR
MECHANICS OF PERFECT ELASTIC BODIES

- $\sigma_{\text {induced }} \leq$ Elastic Limit $\Rightarrow$ Perfect elastic Body
- $\sigma_{\text {induced }}>$ Yield Strength $\Rightarrow$ Perfect Plastic-Body



Bending $\rightarrow$ Two equal Parallel opposite
Pure Bending
$\rightarrow$ ie.

$$
\text { Axial Load }=\text { Shear }=\text { Twisting }=\text { ZERO }
$$

Force Moment
Bending moment $=$ Constant


Torsional $\rightarrow$ Two equal and opposite Parallel eccentric
couple

$$
\begin{aligned}
\text { Axial load }=\text { Shear force }= & \text { Bending }=\text { zero } \\
& \text { Moment }
\end{aligned}
$$

Torsional Moment = Constant

Pure Axial Load

$$
\begin{aligned}
& \sigma_{Q}=\frac{P}{A} ; \delta_{L}=\frac{P L}{A E} \\
& \delta V=\frac{P L}{E}(1-2 u)
\end{aligned}
$$

$$
F O S=\frac{\text { Failure stress }}{\text { Per }} \text { Stress }
$$

## Aimofsom subject $\rightarrow$

Aim of SOM Subject is to derive expressions for Stress, strain and deformation under different loading conditions by using expermentally obtained elastic ptoperties like, Youngs modulus, Poissons ratio.

- Input data for strength of material problem
(I) Load
(ii) dimensions
(III) elastic Ptoperties like $E$ and $\mu$
- Parameters to be determined
(I) Sthess
(II) Strain, by using elastic constants (Properties)
(III) deformations, change in Length, change in cross-sectional dimensions and change in volume
- Assumptions made while deriving SOM equations

1. Perfect elastic member (Stress, sthain With in the elasticregion).
2. Homogeneous and Isotropic material is assumed.
3. Prismatic member is assumed.
4. self weight is neglected.
5. Static Load (gradually applied Load) is assumed.
6. Member is under Static equilibrium condition.

To obtain Safe dimensions in Presence of above mentioned assumptions factor of safety (FOS) should be used in design calcuations.

- Aim of Machine design

Ultimate aim of design is to develop a drawing in such a way that It should Perform its given functionity Satisfactorily. (that is without any failure)

- Stepsuin design of a component:
used

1. Specify the functionality of that component.
2. determination of Loads acting on that component during its functionality.
3. Selection of an appropriate shape.
4. Selection of an appropriate Material.
5. Calculation of dimensions by using strength of Material equation.
$I^{\text {st }}$ Method: By using strength \& rigidity Criterion
II Id method: By using Theory of failure (T.O.F.)
6. selection ofvmanufacturing process detail (i.e. ty Pe of manufacturim, appropriate
Surface finish, Limits and fits).
7. Part drawing should be Prepared for that Component.

$$
G_{\substack{\text { Young's }}}^{E=\frac{\text { Normal stress }}{\text { Longtitudamal strain }} \quad G=\frac{\tau}{\gamma}}
$$

Modulus


```
Strength \(\rightarrow\) Uniaxial
Theories of \(\rightarrow\) Uniaxial, biaxial, triaxial
failure
```

HOMOGENEOUS MATERIAL：－
A material is said to be Homogeneous when it exibits exhibits Same elastic properties at any Point in a given direction （that is elastic Properties are independent of Point）．

## －ISOTROPIC MATERIAL：$\rightarrow$

－A material is said to be isotropic when it exhibits same elastic Properties in any direction at a given Point（i．e．elastic Properties are Independent of direction）
$\rightarrow$ A Material is said to be both Homogeneous and Isotropic when it exhibits same elastic properties at any point and in any direction （ie．elastic Properties are independent of both point and direction）．
$\rightarrow$ Every Homogeneous material need not beanisotopic material． and vice－verssa．but fum materials are both Homogeneous and isotropic
－Anisotropic Material $\rightarrow$（Composite material eg．fiber） A material is said to be Anisotropic when it exhibits direction dependent elastic poperies at a given Point．

保 Tach
＊．Advantage of Anisotropic material
－desirable Property can be achieve as Per requirement． －Higher strength to weight ratio

