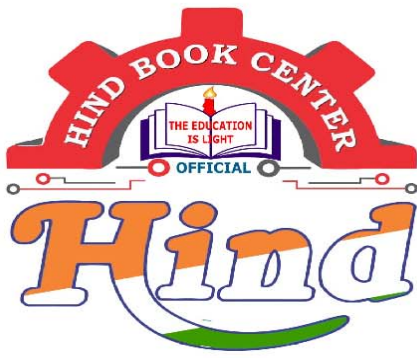




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Structural Analysis

Obj + conv. 1. Deflection of beams

Obj 2. Determinacy & indeterminacy

Obj + conv. 3. Force method of Analysis

conv. 4. Displacement methods of analysis

- slope deflection method
- moment distribution method

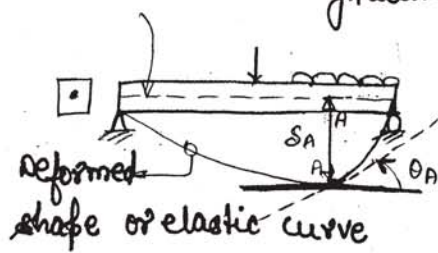
Obj + conv. 5. Truss

Obj 6. Influence line diagram

Obj 7. Matrix methods

Obj 8. Cables and Arches

Chapter 1: Deflection of beams



Internal forces
Tangent to elastic curve at A

Axial force
shear force
BM
TM

S.F & A.F → deformation neglected

In case of beams & frame, deformation is mainly due to BM.

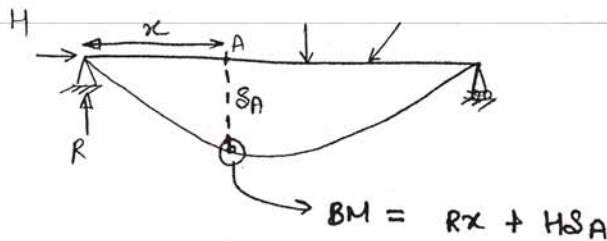
In case of truss deformation is only due to axial force.

θ_A = slope at A

δ_A = Deflection at A

Assumptions & Important points:

- Hooke's law is valid $\Rightarrow$ ^{Homogeneous} linearly elastic condition.
 - Principle of superposition is valid
 - linearly elastic
 - Displacements/deformation are small
- $\Rightarrow$ small displacement theory is valid.



original position → Due to H
no B.M.

but in large displacement

→ Due to H also B.M.
will be there

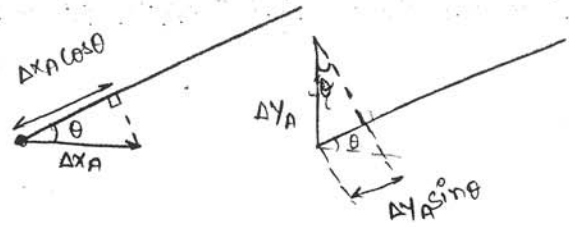
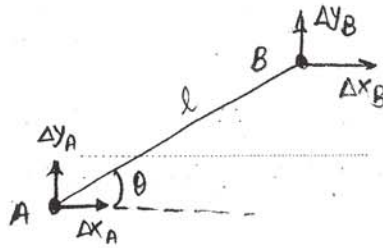
If δ_A is small

$\Rightarrow$ $B.M. = Rx$

→ 1st order analysis

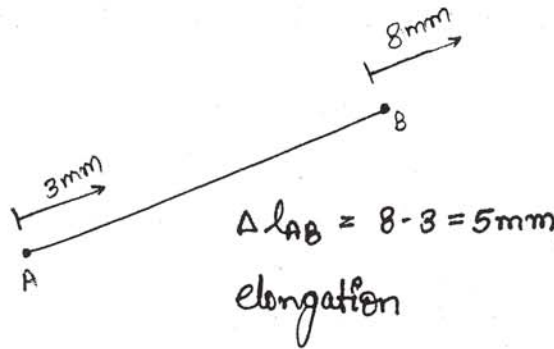
$\Rightarrow$ effect of loading doesn't change due to displacement

• Change in length of member (small displacement theory)

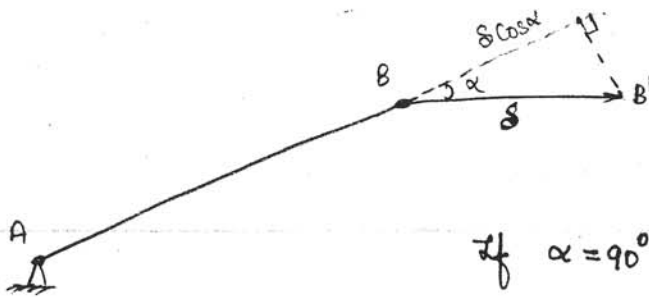


Similarly,
movement of B along AB
 $= \Delta x_B \cos \theta + \Delta y_B \sin \theta$

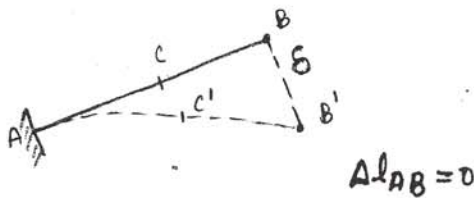
movement of A along AB
 $= \Delta x_A \cos \theta + \Delta y_A \sin \theta$



* change in length of member
 $= [\Delta x_B \cos \theta + \Delta y_B \sin \theta] - [\Delta x_A \cos \theta + \Delta y_A \sin \theta]$



If $\alpha = 90^\circ$, $\Delta l_{AB} = 0$

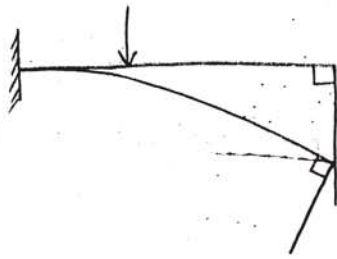


$l_{ACB} = l_{AC'B'}$

* If one joint is restrained then for no change in length of member, other joint members move perpendicular to member.

- In beams & frames joints are rigid.

→ angles b/w members doesn't change due to deformation.



- Section



sagging BM



curvature



hogging BM



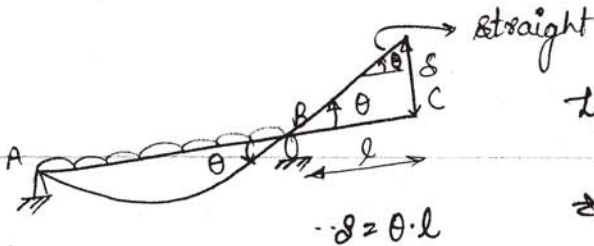
curvature

$$\text{Curvature} = \frac{1}{R} = \frac{M}{EI}$$

M = Bending moment

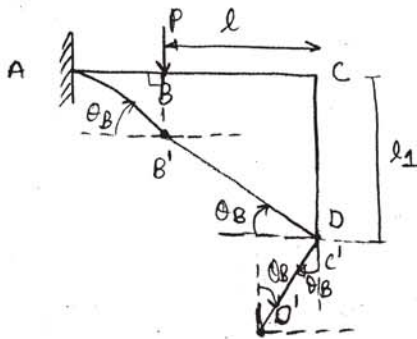
EI = Flexural rigidity

If $M=0 \rightarrow$ ————— No curvature



In BC portion, $BM=0$

→ In BC portion, ~~curv~~ curvature = 0

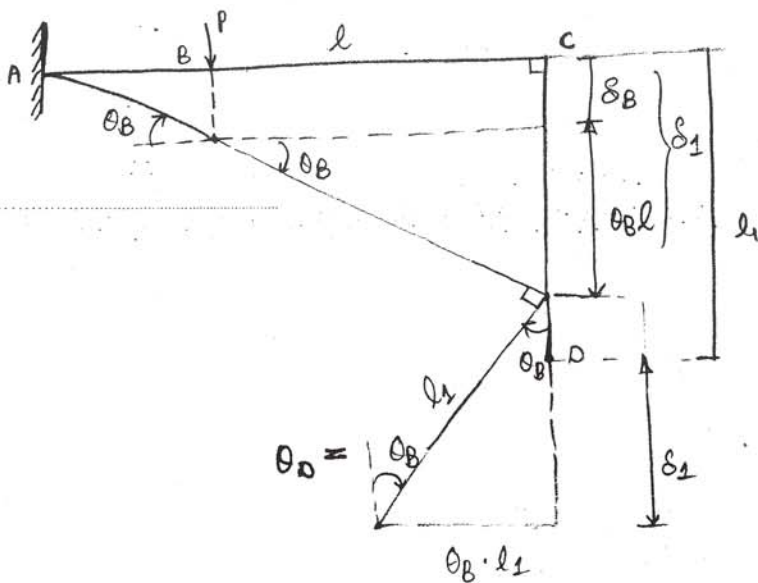


BM in part BCD is zero

→ curvature in part BCD = 0

CC' will be $\perp$ cular to AC
so that length of AC doesn't change.

$$l_{ABC} = l_{AB'C'}$$



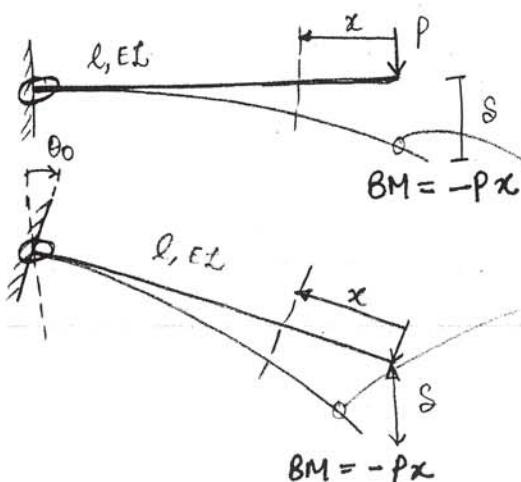
for no change in length of CD.

$$\Delta D_H = \theta_B l_1 \leftarrow$$

$$\Delta D_V = \delta_B + \theta_B l_1 \downarrow$$

$$\theta_D \downarrow = \theta_B$$

change in length of members = 0 $\rightarrow$ assumed in beams & frame i.e. axially rigid members.

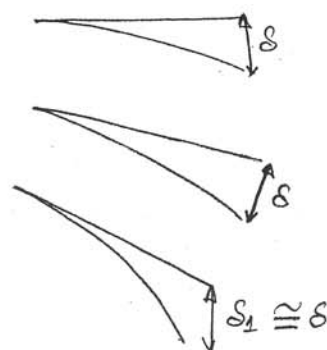


$$\text{Curvature} = \frac{1}{R} = \frac{M}{EI}$$

Curvature same

BM same $\rightarrow$ curvature same

$\rightarrow$ but slope & deflection may not be same

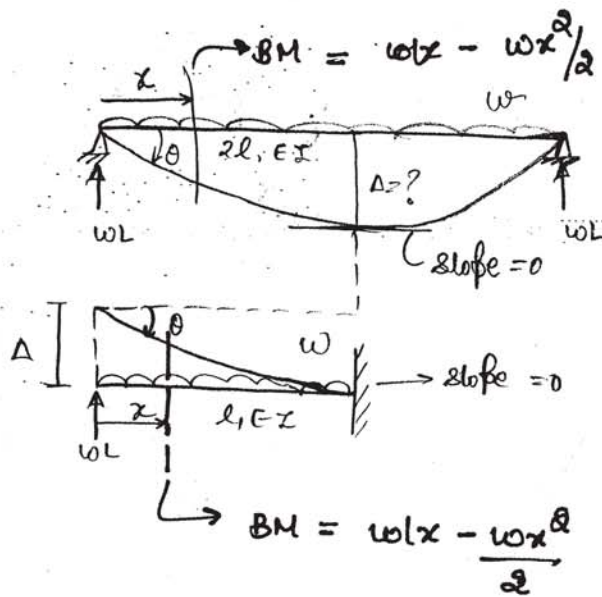


If $\frac{B \cdot M}{EI}$ throughout the length of beam is same in 2 beams then curvature of deflected shape will be same but for slope & deflection to be also same, at one corresponding location slope in the 2 beams must be same.

about this point

shape same

Example :



same $\frac{M}{EI}$

same slope at
one corresponding
location

↓
same δ & θ

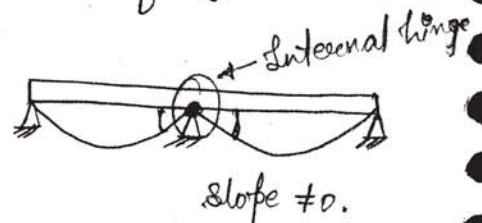
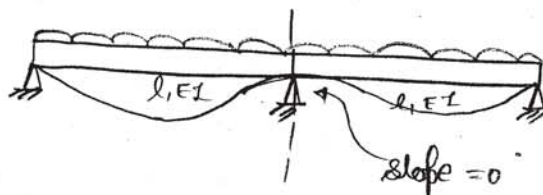


6. Under symmetrical loading) BMD is symmetrical, SFD is antisymmetric

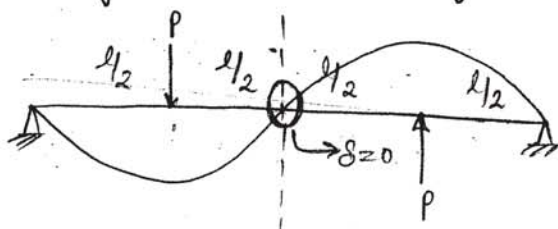
c) deflected shape $\rightarrow$ symmetrical

d) SF at axis of symmetry $= 0$

e) slope at axis of symmetry is zero if there
... is no internal hinge at axis of symmetry.



for antisymmetric loading

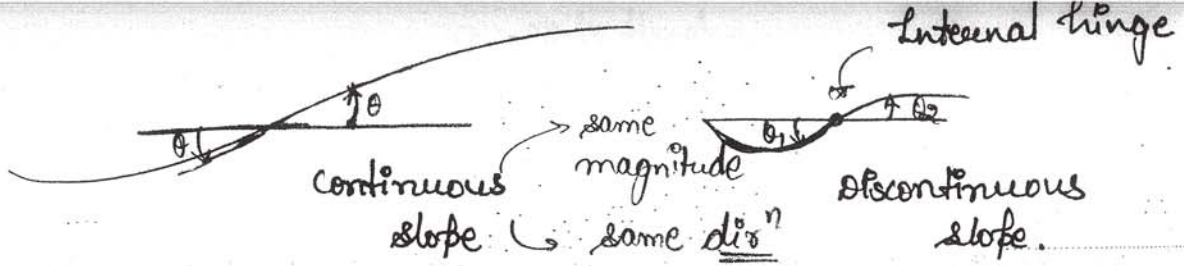


• BM antisymmetric

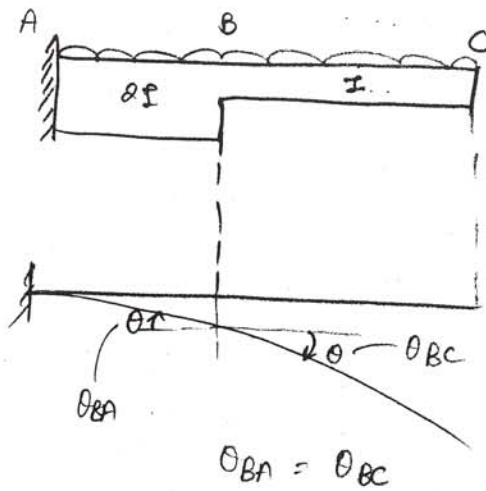
• SFD $\rightarrow$ symmetric

• deflected - Antisymmetric
shape

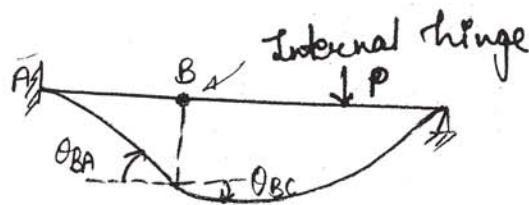
• At axis of antisymmetry deflection $= 0$



7.

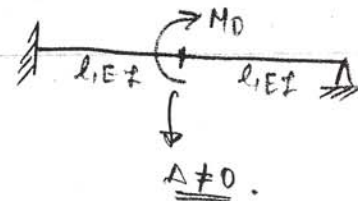
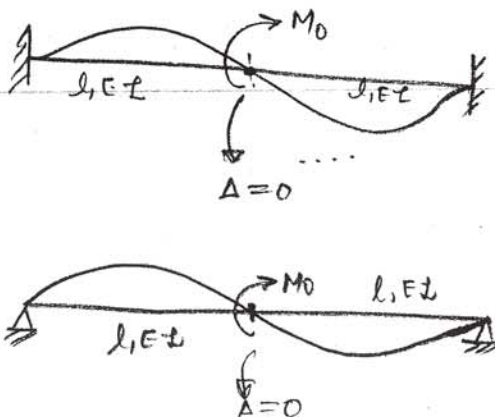


At one section, slope just to the left & slope just to the right will be same except when there is internal hinge at that location.



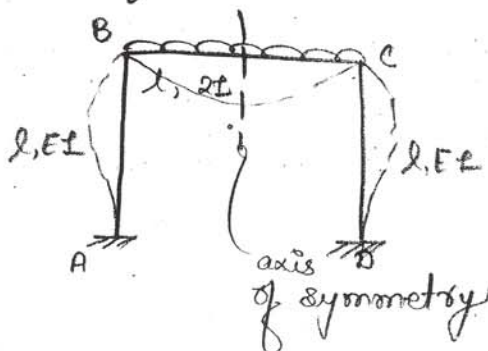
$$\theta_{BA} \neq \theta_{BC}$$

8.



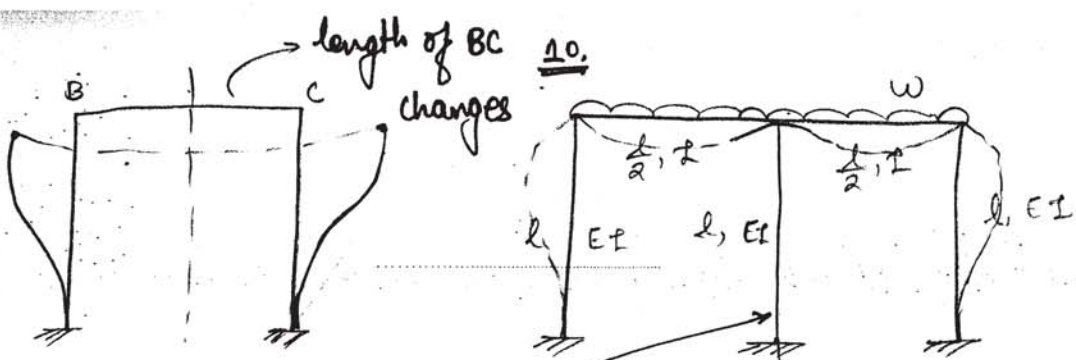
9.

In case of symmetrical frames, joints do not sway



if axial rigidity of members exist

Joint (B) & (C) will not move from its position if axial rigidity of members is assumed.

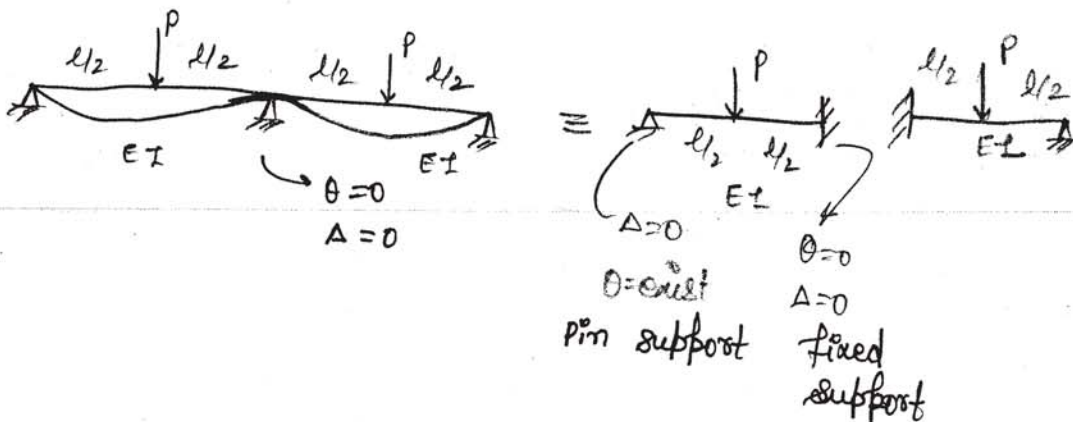
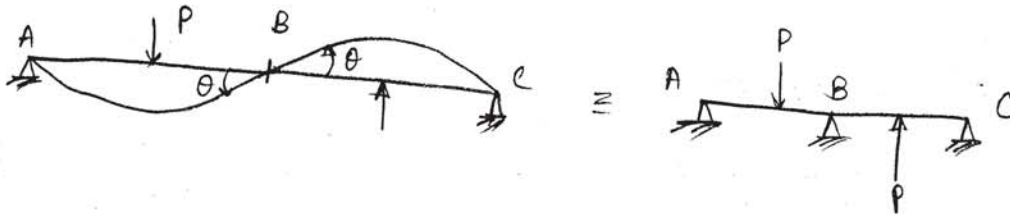


will
not bend
⇒ $BM = 0$

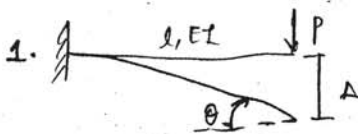
Axis of symmetry passes through
a column.

↳ This column will not bend
& will not carry any BM

11.

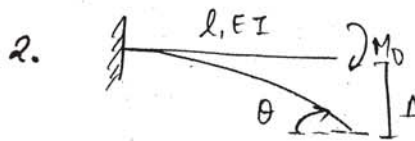


Some standard results :-



$$\Delta = \frac{Pl^3}{3EI}$$

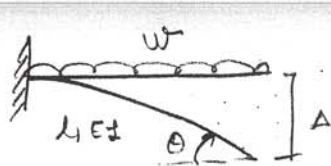
$$\theta = \frac{Pl^2}{2EI}$$



$$\Delta = \frac{M_0 l^2}{2EI}$$

$$\theta = \frac{M_0 l}{EI}$$

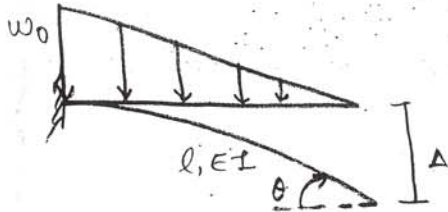
3.



$$\Delta = \frac{wl^4}{8EI}$$

$$\theta = \frac{wl^3}{6EI}$$

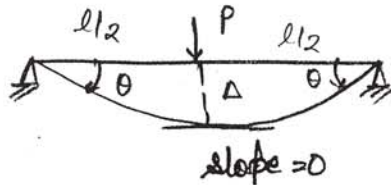
4.



$$\Delta = \frac{w_0 l^4}{80EI}$$

$$\theta = \frac{w_0 l^3}{24EI}$$

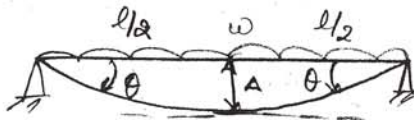
5.



$$\Delta = \frac{Pl^3}{48EI}$$

$$\theta = \frac{Pl^2}{16EI}$$

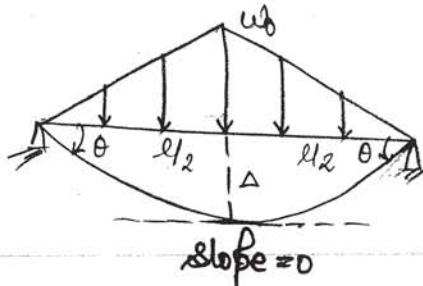
6.



$$\Delta = \frac{5}{384} \frac{wl^4}{EI}$$

$$\theta = \frac{wl^3}{24EI}$$

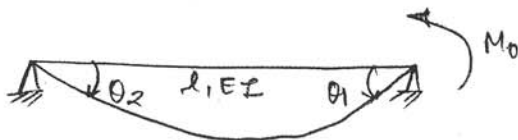
7.



$$\Delta = \frac{w_0 l^4}{120EI}$$

$$\theta = \frac{5w_0 l^3}{192EI}$$

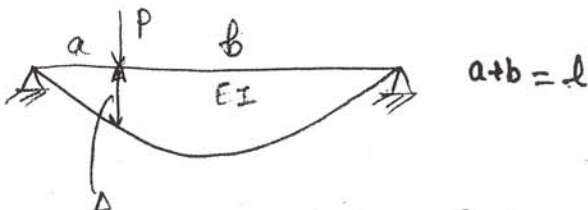
8.



$$\theta_1 = \frac{M_0 l}{3EI}$$

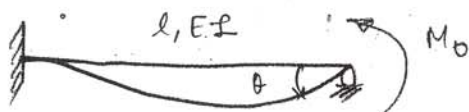
$$\theta_2 = \frac{M_0 l}{6EI}$$

9.



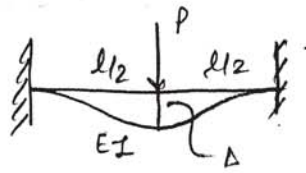
$$\Delta = \frac{P a^2 b^2}{3EI l}$$

10.



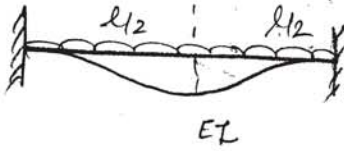
$$\theta = \frac{M_0 l}{4EI}$$

11.



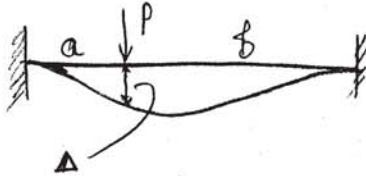
$$\Delta = \frac{1}{4} \left[\frac{PL^3}{48EI} \right]$$

12.



$$\Delta = \frac{1}{5} \left[\frac{5}{384} \frac{wL^4}{EI} \right]$$

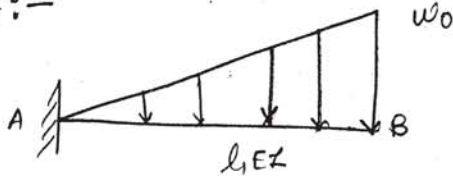
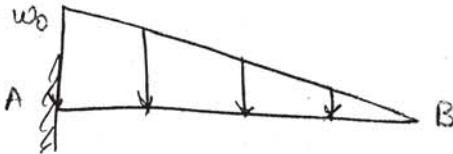
13.



$$a+b=L$$

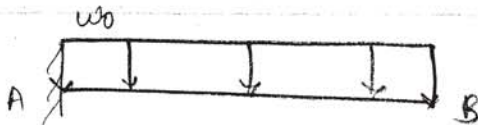
$$\Delta = \frac{Pa^3b^3}{3EI L^8}$$

Que:-

find θ_B & A_B 

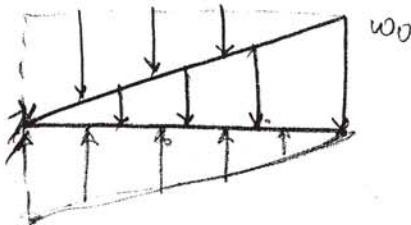
$$\theta_B = \frac{w_0 L^3}{24EI}$$

$$A_B = \frac{w_0 L^4}{30EI}$$

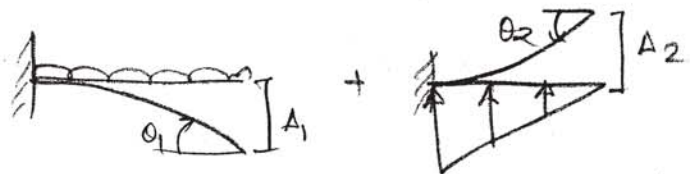


$$\theta_B = \frac{w_0 L^3}{6EI}$$

$$A_B = \frac{w_0 L^4}{8EI}$$



=



$$\Delta = \Delta_1 - \Delta_2$$

$$\theta = \theta_1 - \theta_2$$

$$\Delta = \frac{w_0 L^4}{8EI} - \frac{w_0 L^4}{30EI} = \frac{11}{40} \frac{w_0 L^4}{EI} \downarrow$$

$$\theta = \frac{w_0 L^3}{6EI} - \frac{w_0 L^3}{24EI}$$

$$\theta = \frac{w_0 L^3}{8EI} \downarrow$$